

CMGT 235 – Electrical and Mechanical Systems

Department of Construction Management ☻ California State University, Chico

Exam #1 [100 points]

You may work together as a group or individually. Every student SHALL complete their own answer sheet.

| |
|-----------------------|
| Name: Solution |
| Name: |
| Name: |

15 pts 1. A 6 in x 6 in x 3 in ice cube is held in a freezer at 0°F. How much heat is required to turn the ice cube to steam?

Weight of Ice Cube

$$M = (0.5 \text{ ft} \times 0.5 \text{ ft} \times 0.25 \text{ ft}) \times 62.41 \text{ lb/ft}^3 = 3.9 \text{ lb}$$



Sensible Heat 0°F to 32°F solid

$$Q = M \times C \times \Delta T = 3.9 \text{ lb} \times 1 \text{ Btu/lb}^\circ\text{F} \times (32^\circ\text{F} - 0^\circ\text{F}) = 125 \text{ Btu}$$



Latent Heat of Fusion 32°F to Melted

$$Q = 144 \text{ Btu/lb} \times 3.9 \text{ lb} = 562 \text{ Btu}$$



Sensible Heat Melted @32°F to 212°F

$$Q = M \times C \times \Delta T = 3.9 \text{ lb} \times 1 \text{ Btu/lb}^\circ\text{F} \times (212^\circ\text{F} - 32^\circ\text{F}) = 702 \text{ Btu}$$



Latent Heat of Vaporization 212°F to Steam

$$Q = 970 \text{ Btu/lb} \times 3.9 \text{ lb} = 3783 \text{ Btu}$$

$$\text{Total Heat (Enthalpy)} = 125 \text{ Btu} + 562 \text{ Btu} + 702 \text{ Btu} + 3783 \text{ BTU} = 5,172 \text{ Btu}$$

35 pts 2. For the tiny home plan shown the inside temperature is 72°F and the outside temperature is -15°F.

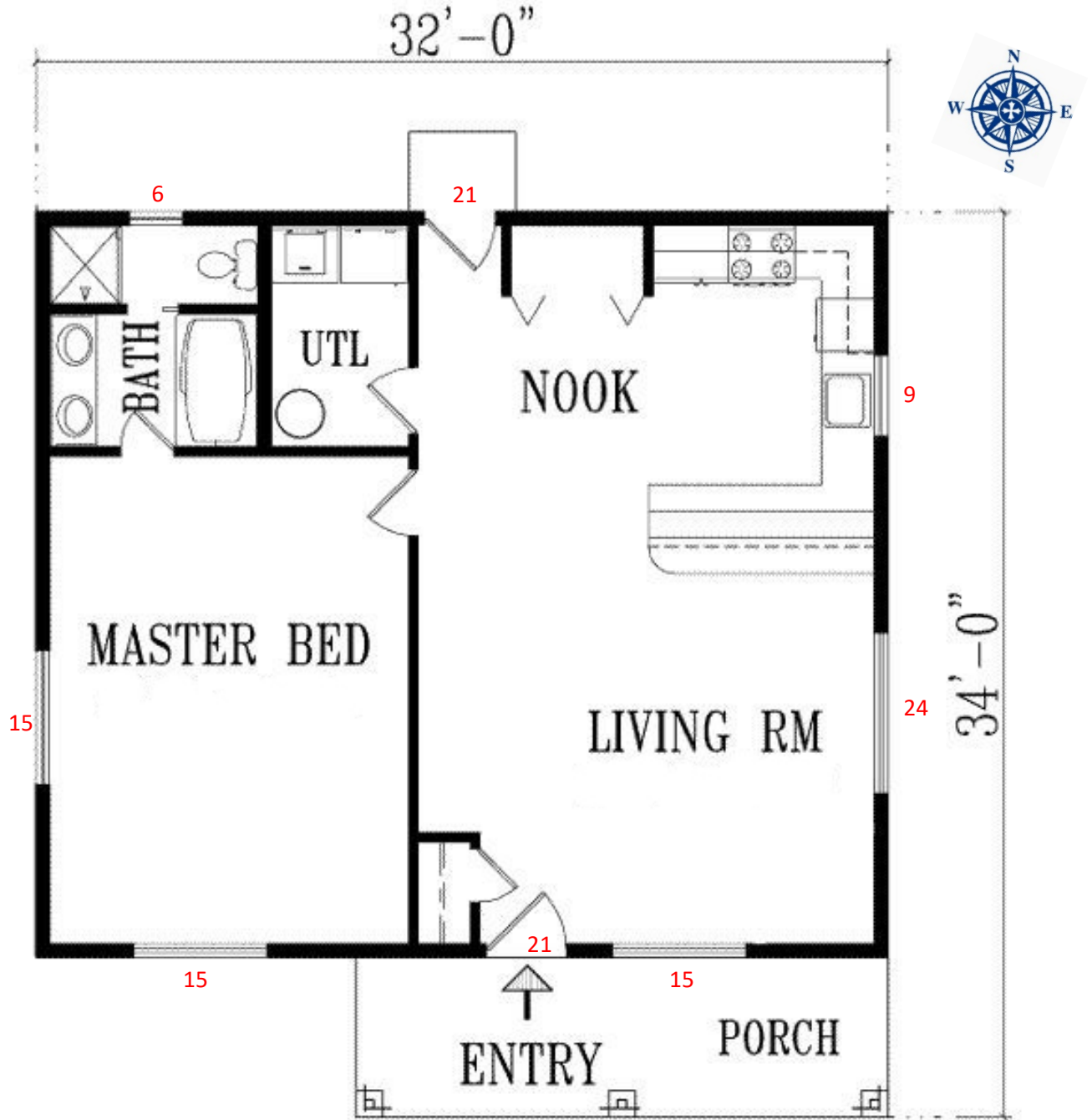
Specifications

Window Sizes

| | |
|----------------|-----------------------------|
| Bathroom | 3 ft x 2 ft |
| Nook | 3 ft x 3 ft |
| Living Room | 6 ft x 4 ft and 5 ft x 3 ft |
| Master Bedroom | 5 ft x 3 ft E |

| | | |
|----------|---------------------------------------|-------|
| Walls | R-19 (6" insulation) | 0.053 |
| Ceilings | R-30 (10" insulation) | 0.033 |
| Windows | R-3.13 | 0.319 |
| Doors | R-3.70 | 0.270 |
| Floor | SOG (2 in thick edge insulation, R=5) | |

| | |
|----------------|----------------------|
| Door | 21 ft ² E |
| Ceiling Height | 8 ft |
| Wood Porch | 4 ft x 18 ft |



Use 3-decimals for all U-factors. Round all calculations to whole numbers.

A. Determine the total heat loss due to transmission.

$$\text{Gross Wall Area} = 2 \times 32 \text{ ft} \times 8 \text{ ft} + 2 \times 30 \text{ ft} \times 8 \text{ ft} = 512 \text{ ft}^2 + 480 \text{ ft}^2 = 992 \text{ ft}^2$$

$$\text{Window Area} = 6 \text{ ft}^2 + 9 \text{ ft}^2 + 24 \text{ ft}^2 + 15 \text{ ft}^2 + 15 \text{ ft}^2 + 15 \text{ ft}^2 = 84 \text{ ft}^2$$

$$\text{Door Area} = 2 \times 21 \text{ ft}^2 = 42 \text{ ft}^2$$

$$\text{Net Wall Area} = 992 \text{ ft}^2 - 84 \text{ ft}^2 - 42 \text{ ft}^2 = 866 \text{ ft}^2$$

$$q_{\text{transmission}} = U \times A \times \Delta T$$

$$q_{\text{Walls}} = 0.053 \times 866 \text{ ft}^2 \times (72^\circ\text{F} - -15^\circ\text{F}) = 3993 \text{ BTUH}$$

$$q_{\text{Windows}} = 0.319 \times 84 \text{ ft}^2 \times (72^\circ\text{F} - -15^\circ\text{F}) = 2331 \text{ BTUH}$$

$$q_{\text{Doors}} = 0.270 \times 42 \text{ ft}^2 \times (72^\circ\text{F} - -15^\circ\text{F}) = 987 \text{ BTUH}$$

$$q_{\text{Ceiling}} = 0.033 \times (32 \text{ ft} \times 30 \text{ ft}) \times (72^\circ\text{F} - -15^\circ\text{F}) = 2756 \text{ BTUH}$$

$$q_{\text{Slab}} = U_f \times L = 45 \times (2 \times 32 \text{ ft} + 2 \times 30 \text{ ft}) = 45 \times 124 \text{ ft} = 5580 \text{ BTUH}$$

$$\text{Total Heat Loss Due to Transmission} = 15,647 \text{ BTUH}$$

B. Determine the heat loss due to infiltration for an ACH = 1.2

$$q_{\text{infiltration}} = 0.018 \times V \times \text{ACH} \times \Delta T$$

$$= 0.018 \times (32 \text{ ft} \times 30 \text{ ft} \times 8 \text{ ft}) \times 1.2 \times (72^\circ\text{F} - -15^\circ\text{F})$$

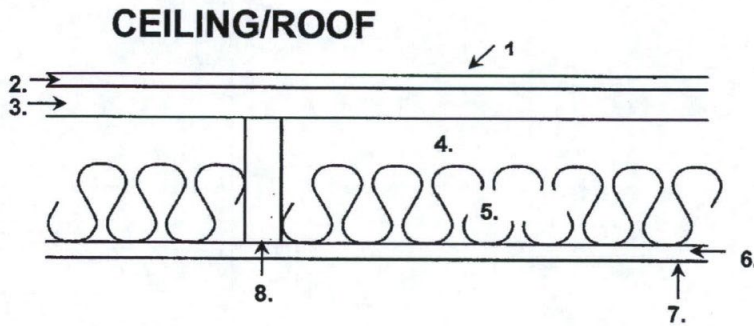
$$= 0.018 \times 7680 \text{ ft}^3 \times 1.2 \times (72^\circ\text{F} - -15^\circ\text{F})$$

$$= 14,432 \text{ BTUH}$$

C. Determine the Total Heat Loss.

$$q_{\text{total}} = q_{\text{transmission}} + q_{\text{infiltration}} = 15,647 + 14,432 = 30,079 \text{ BTUH}$$

- 25 pts 3. Calculate the total heat loss during a 24-hour period for a flat roof 55 ft X 70 ft. The roof is constructed per the detail below. The inside temperature is 70 °F and the outside temperature is 52 °F. Assume winter conditions. Use 2-decimals for R Values and 3-decimals for U-Factors. Round answer to a whole number.



| | | R (Between joist) | R (At joist) |
|--------------------|---|-------------------|--------------|
| 1. | Air film outside | 0.17 | 0.17 |
| 2. | 3/8 in. Built-up roofing | 0.33 | 0.33 |
| 3. | 5/8 in. Plywood Sheathing | 0.77 | 0.77 |
| 4. | 1 ½ in. Air space | 1.00 | --- |
| 5. | R-13 Fiberglass Batt Insulation | 13.00 | --- |
| 6. | 5/8 in. Gypsum board | 0.56 | 0.56 |
| 7. | Air film inside | 0.68 | 0.68 |
| 8. | Nominal 2-in x 12-in Doug Fir Joist @ 24 in. o.c. (1.5" x 11.25") R = 1.00 | --- | 11.25 |
| R _{Total} | | 16.51 | 13.76 |

Determine the average U-Factor for the ceiling



$$R_{AVG} = (1.5''/24'') \times 13.76 + (22.5''/24'') \times 16.51 = 0.86 + 15.48 = 16.34$$

$$U_{AVG} = 1 / R_{AVG} = 1 / 16.34 = 0.061$$

Total Heat Loss

$$Q = U \times A \times \Delta T \times 24 \text{ hr} = 0.061 \times (55 \text{ ft} \times 70 \text{ ft}) \times (70^\circ\text{F} - 52^\circ\text{F}) \times 24 = 101,455 \text{ BTU}$$

Psychrometric Chart – See Chart on next page

- 10 pts 4. Given the ambient temperature is 70°F measured by a dry bulb thermometer and 60°F measured by a wet bulb thermometer, what is the relative humidity?

$$RH = 56\%$$

Is this point an acceptable temperature and humidity for personal comfort all year for people in the USA?

No. Falls into the “Winter Comfort Zone” but not the “Summer Comfort Zone.”

- 15 pts 5. A house is 4500 ft² and has 12 ft ceilings. For comfort, the homeowner specifies 0.3 changes of air per hour. The outside air temperature is 90°F dry bulb and 73.5° wet bulb. The air indoors is 75°F dry bulb 50% relative humidity. What is the amount of cooling required to provide the fresh air?

See page 6 - About the Psychrometric Chart (Power knot)

$$\text{Volume} = 4500 \text{ ft}^2 \times 12 \text{ ft} = 54,000 \text{ ft}^3$$

$$\text{CFM} = 54,000 \text{ ft}^3 \times 0.3 \text{ x hr} / 60 \text{ min} = 16,200 \text{ ft}^3/\text{hr} \text{ x hr} / 60 \text{ min} = 270 \text{ CFM}$$

From Psychrometric Chart

Outdoor Air

$$DB = 90^\circ\text{F}$$

$$WB = 73.5^\circ\text{F}$$

$$\text{Enthalpy} = 37.2 \text{ BTU} / \text{lb DA}$$

$$\text{Specific Volume} = 14.2 \text{ ft}^3 / \text{lb}$$

Indoor Air

$$DB = 75^\circ\text{F}$$

$$RH = 50\%$$

$$\text{Enthalpy} = 28.4 \text{ BTU} / \text{lb DA}$$

$$\text{Specific Volume} = 13.7 \text{ ft}^3 / \text{lb}$$

$$\text{Energy of Incoming Air} = (16,200 \text{ ft}^3 / \text{hr} \times 37.2 \text{ BTU} / \text{lb DA}) / 14.2 \text{ ft}^3 / \text{lb} = 42,439 \text{ BTUH}$$

$$\text{Energy of Indoor Air} = (16,200 \text{ ft}^3 / \text{hr} \times 28.4 \text{ BTU} / \text{lb DA}) / 13.7 \text{ ft}^3 / \text{lb} = 33,583 \text{ BTUH}$$

$$\text{Heat Difference} = 42,439 \text{ BTUH} - 33,583 \text{ BTUH} = 8,856 \text{ BTUH}$$

$$\text{Cooling Needed} = 8,856 \text{ BTUH} / 12,000 \text{ BTU} = 0.738 \text{ ton or about } 0.75 \text{ ton (3/4-ton AC Unit)}$$

Psychrometric chart at sea level (760 mm Hg)
US measurement units

